

How to model offset wagon loads in Vampire

Vampire User Event - 14th June 2022

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How are static loads normally calculated in Vampire models?

- > Static loads in weight-bearing elements are automatically calculated by Vampire
- > If there are parallel load paths, load distributed according to stiffness
- Static loads not automatically used for normal loads on friction elements
- PRELOAD keyword in bumpstop adjusts characteristic but does not normally change the actual static load in the element









When wagon assembled, sidebearers contact first and load up to 16 kN each before centre pivot comes into contact





- Sidebearer spring vertical stiffness = 0.285 MN/m, so 1.14 MN/m per bogie
- Centre pivot stiffness could be 100 MN/m
- Assume 200 kN pivot load
- Actual loads are 16 kN per sidebearer and 168 kN on pivot
- Calculated according to stiffness only:
 - Load per sidebearer = 200 x 0.57 / (1.14 + 100) = 1.127 kN
 - Load on pivot = 200 x 100 / (1.14 + 100) = 197.746 kN





Model sidebearer bumpstop characteristic as follows (assuming 12 mm gap):

DISPLACEMENT FORCE	-22.00 -606.84	-12.00 -6.84	0.00	28.07 16.00	$\begin{array}{c} 100.00\\ 16.00 \end{array}$
This is equivalent f	to:				
DISPLACEMENT FORCE PRELOAD	-50.07 -622.84 16.0	-40.07 -22.84	-28.07 -16.00	0.00	71.93 0.00

This means that if 16 kN is removed from the sidebearer it will lose contact, but there is only 1.127 kN on it, so it will reach 14.873 kN TENSION before releasing













> To force the preload to be 16 kN add STIC OVERRIDE:

DISPLACEMENT	-50.07	-40.07	-28.07	0.00	71.93
FORCE	-622.84	-22.84	-16.00	0.00	0.00
PRELOAD	16.0				
STATIC	OVERRIDE				

Unless the centre pivot is also adjusted this will lead to an imbalance in the model as the sidebearers will carry 16 kN each but the centre pivot will still carry 197.746 kN so the total load carried will be 229.746 kN





- > Assume centre pivot is modelled with a 100 MN/m STIFFNESS element
- Displacement under 168 kN load would be 168 / 100 = 1.68 mm or 0.00168 m
- \blacktriangleright If free height is 0.100 m then installed height would be 0.100 0.00168 = 0.09832 m

Model like this:

STIFFNESS	100.00		
FREELENGTH	0.100		
STATIC **	OVERRIDE		
POSITION	8.000	0.000	0.79832
	8.000	0.000	0.700





- Makes very little difference to model behaviour
- Friction static normal loads are defined separately
- > Does affect total load on each element- check using BSTnn and KSTnn output mnemonics
- Total bumpstop load:

BST01+B01

Total stiffness load:

KST01+K01







Why are we concerned about static loads in models?

- Parallel load paths such as three-axle bogies
- Load distribution affected by body or bogie twist
- Loadings with offset centres of gravity, especially with non-linear and multi-stage suspensions where elements come into contact at different loads
- Example- new offset container load cases in GMRT2141 issue 4.1





- Following Camden Road derailment in 2013 three offset container loading cases introduced to GMRT2141 Issue 4 (now 4.1) for delta Q/Q (and potentially Y/Q) assessment
- Case a- lateral offset

Case b- combined offset 1

Case c- combined offset 2

1.00	1.00
1.38	1.38
1.00	1.35
1.20	1.62
1.00	3.00
1.10	3.30





How to model these accurately with correct static loads in sidebearers and primary tare















- Body could roll on pivots, closing sidebearer gaps on one side, possibly as far as metal contact
- Bogie could drop and roll, increasing loads in Lenoir links and friction plungers and could drop and roll enough to contact laden springs on one or both sides
- The static loading could be precalculated manually but this seems very tedious when Vampire can do it for you!





Create a tare model including elements with defined static loads

Use parameters such as:

d1	=	0.0000	m	Drop at end 1
d 2		0.0000	m	Drop at end 2
r1		0.00000	rad	Bogie roll at end 1
r2	=	0.00000	rad	Bogie roll at end 2



Vampire Pro

Model shear springs with FREELENGTH and STATIC OVERRIDE:

rings						
0.500	0.500	0.800	0.001	0.001	0.	001
0.300						
OVERRIDE						
X	1.000	0.56319-	d1-1*r1	0.300	2	4
X	-1.000	0.56319-	d1+1*r1	0.300	2	4
orings						
0.300	0.300	1.500	0.001	0.001	0.	001
0.255						
OVERRIDE						
X	1.000	0.56319-	d1-1*r1	0.300	2	4
X	-1.000	0.56319-	d1+1*r1	0.300	2	4
	rings 0.500 0.300 OVERRIDE X x orings 0.300 0.255 OVERRIDE X X	rings 0.500 0.500 0.300 OVERRIDE x 1.000 x -1.000 orings 0.300 0.300 0.255 OVERRIDE x 1.000 x -1.000	rings 0.500 0.500 0.800 0.300 OVERRIDE x 1.000 0.56319- x -1.000 0.56319- orings 0.300 0.300 1.500 0.255 OVERRIDE x 1.000 0.56319- x -1.000 0.56319-	rings 0.500 0.500 0.800 0.001 0.300 OVERRIDE x 1.000 0.56319-d1-1*r1 x -1.000 0.56319-d1+1*r1 orings 0.300 0.300 1.500 0.001 0.255 OVERRIDE x 1.000 0.56319-d1-1*r1 x -1.000 0.56319-d1-1*r1 x -1.000 0.56319-d1+1*r1	rings 0.500 0.500 0.800 0.001 0.001 0.300 OVERRIDE x 1.000 0.56319-d1-1*r1 0.300 x -1.000 0.56319-d1+1*r1 0.300 orings 0.300 0.300 1.500 0.001 0.001 0.255 OVERRIDE x 1.000 0.56319-d1-1*r1 0.300 x -1.000 0.56319-d1+1*r1 0.300	rings 0.500 0.500 0.800 0.001 0.001 0.00 0.300 OVERRIDE x 1.000 0.56319-d1-1*r1 0.300 2 x -1.000 0.56319-d1+1*r1 0.300 2 orings 0.300 0.300 1.500 0.001 0.001 0.00 0.255 OVERRIDE x 1.000 0.56319-d1-1*r1 0.300 2 x -1.000 0.56319-d1+1*r1 0.300 2





> With d1 = r1 = 0, tare spring load will be:

1000 x 0.8 x (0.3 - (0.56319 - 0.3) = 800 x 0.03681 = 29.448 kN

Laden spring load will be 0 as installed height is less than free height







- Create a 'setup' model by setting friction to very low values (Vampire often doesn't like it set to zero)
- > Apply offset container loads to setup model using static program
- > All loads will be carried by springs, not friction





STATIC LOAD EXAMPLE

```
..\D01 Vehicle models\Example_setup
**
```

***STATIC**

DQ	0.00001
STAGE 1	
** Apply	load
F01Z	38.001
F01T	18.210
F01P	-101.221
**	





*OUTPUT

Primary drop, bogie 1
D02Z-(D04Z+D05Z)/2
Primary drop, bogie 2
D03Z-(D06Z+D07Z)/2
Roll, bogie 1
D02T-(D04T+D05T)/2
Roll, bogie 2
D03T-(D06T+D07T)/2
*

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m

m

rad

rad

Read results from STATIC section of lis file:

```
*STATIC DATA:-
CH 1 Primary drop, bogie 1 m
    1.049E-02
CH 2 Primary drop, bogie 2 m
    1.271E-03
CH 3 Roll, bogie 1 rad
    2.842E-03
CH 4 Roll, bogie 2 rad
    2.834E-03
```





Use the results to set the parameters in the laden vehicle model:

d1	=	0.01049	m	Drop at end 1
d2	=	0.00127	m	Drop at end 2
r1	=	0.00284	rad	Bogie roll at end 1
r2	=	0.00283	rad	Bogie roll at end 2

> The example spring loads will then be as follows:

Tare right 1000 x 0.8 x (0.3 - (0.56319 - 0.01049 - 0.00284 - 0.3) = 40.112 kN Tare left 1000 x 0.8 x (0.3 - (0.56319 - 0.01049 + 0.00284 - 0.3) = 35.568 kN Laden right 1000 x 1.5 x (0.255 - (0.56319 - 0.01049 - 0.00284 - 0.3) = 7.710 kN Laden left 0 as height is greater than free height





> For a full Y series bogie model need to define the following:

- Static bogie and plunger drops
- Static body and bogie rolls
- > Axlebox pitch angles (because of Lenoir links), both bogies, both sides
- Lenoir link preloads, both bogies, both sides
- Friction normal loads (plungers), both bogies, both sides
- Sidebearer and centre pivot loads





- Also need loads in tare and laden springs in order to calculate their shear stiffnesses as these are load-dependent
- Can use INCLUDE files to add these parameters to the model





** EXAMPLE

** ******

***PARAMETER**

d1	=	0.01049	m	Bogie 1	drop	
d2	=	0.00127	m	Bogie 2	drop	
p1	=	0.01022	m	Bogie 1	plunger	drop
p2	=	0.00124	m	Bogie 2	2 plunger	drop
r1	=	0.00284	rad	Bogie 1	roll	
r2	=	0.00283	rad	Bogie 2	roll	
rb	=	0.01110	rad	Body ro	511	





a1R =	-0.00053	rad	Axlebox	1	right pitch
a1L =	-0.00030	rad	Axlebox	1	left pitch
$a^2R =$	0.00052	rad	Axlebox	2	right pitch
a2L =	0.00030	rad	Axlebox	2	left pitch
a3R =	-0.00016	rad	Axlebox	3	right pitch
a3L =	0.00006	rad	Axlebox	3	left pitch
a4R =	0.00016	rad	Axlebox	4	right pitch
a4L =	-0.00006	rad	Axlebox	4	left pitch

*





** EXAMPLE	Ξ *									
*PARAMETER	R *									
*	k0801 k0802	=	0.367 0.362	MN/m MN/m	Bogie Bogie	1 2	tare tare	spring spring	shear shear	stiffness stiffness
** EXAMPLE	E *									
*PARAMETER	PARAMETER									
	k0901 k0902	=	0.347 0.347	MN/m MN/m	Bogie Bogie	1 2	1aden 1aden	spring spring	shear shear	stiffness stiffness



Can even add container load using an INCLUDE file:







	DISPLACEMENT FORCE PRELOAD	-10.000 -600.000 10.052	0.000 0.000	100.000 0.000	
в 73	POSITION	X+D*5.992	D*1.13	1.045-1.31*d1+0.31*d2-1.13*rb	20
		X+D*5.992	D*1.13	0.945-1.31*d1+0.31*d2-1.13*rb	1
	DISPLACEMENT	-10.000	0.000	100.000	
	FORCE	-600.000	0.000	0.000	
	PRELOAD	10.052			
в 74	POSITION	X+D*5.992	D*-1.13	1.045-1.31*d1+0.31*d2+1.13*rb	20
		X+D*5.992	D*-1.13	0.945-1.31*d1+0.31*d2+1.13*rb	1
	DISPLACEMENT	-10.000	0.000	100.000	
	FORCE	-600.000	0.000	0.000	
	PRELOAD	10.052			
в 75	POSITION	X+D*-5.992	D*1.13	1.045+0.31*d1-1.31*d2-1.13*rb	20
		X+D*-5.992	D*1.13	0.945+0.31*d1-1.31*d2-1.13*rb	1
	DISPLACEMENT	-10.000	0.000	100.000	
	FORCE	-600.000	0.000	0.000	
	PRELOAD	10.052			
В 76	POSITION	X+D*-5.992	D*-1.13	1.045+0.31*d1-1.31*d2+1.13*rb	20
		X+D*-5.992	D*-1.13	0.945+0.31*d1-1.31*d2+1.13*rb	1



*BUMPSTOP



*BUSH ****

P 13 P 14 P 15 P 16	STIFFNESS DAMPING POSITION POSITION POSITION POSITION	50.000 0.045 X+D*5.992 X+D*5.992 X+D*-5.992 X+D*-5.992	50.000 0.045 D*1.13 D*-1.13 2 D*1.13 2 D*1.13	0.001 0.045 0.995 0.995 0.995	0.000 0.000 5-1.31*d1 5-1.31*d1 5+0.31*d1 5+0.31*d1	0.000 0.000 +0.31*d2-1 +0.31*d2+1 -1.31*d2-1 -1.31*d2+1	0.000 0.000 L.13*rb L.13*rb L.13*rb L.13*rb	1 1 1 1
P 16	POSITION	X+D*-5.992	2 D*-1.13	0.99	5+0.31*d1	-1.31*d2+1	L.13*rb	1



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Reference the INCLUDE files in the model like this:

*INCLUDE ******* FILE Example part 1.VAMParam

**

Check the model and check the loads in the .slc file to make sure they are as intended and that they give the right masses for every mass and wheelset in the model





How can the process be automated?

- Create and check the tare model and create the low friction 'setup' version
- Create a macro-enabled Excel workbook
- > Define loading in terms of container size, position and mass properties
- > Excel can create and save the static run file and read the results from the lis file
- Excel can create and save the INCLUDE files
- Excel can edit the tare vehicle model to create the laden model, just by changing the title and references to INCLUDE files







- > In some situations it is important to define the static loads in a Vampire vehicle model
- It is important to check the static balance when this is done as it is possible to create models that do not balance
- > The process can be automated, but this requires extreme care
- > A lot to take in here- I am happy to answer questions later if needed!





Thank you

